

# Update in the microchannel cooling simulations of DEPFET vertex detector for a future linear e+ e- collider

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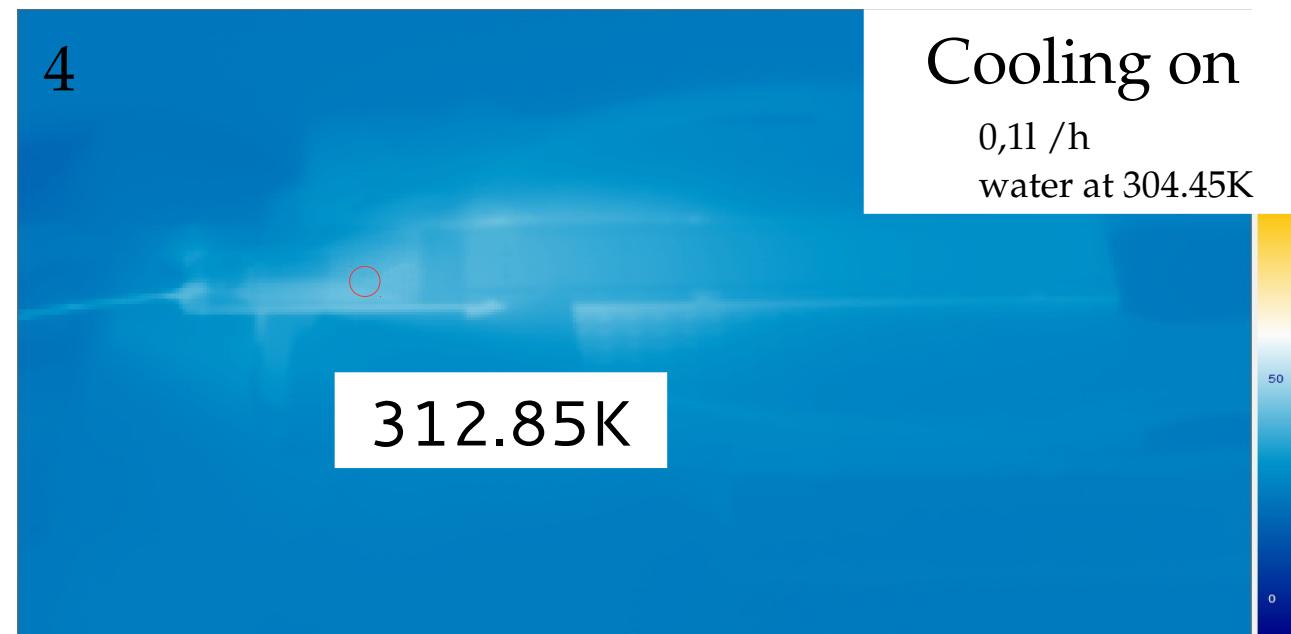
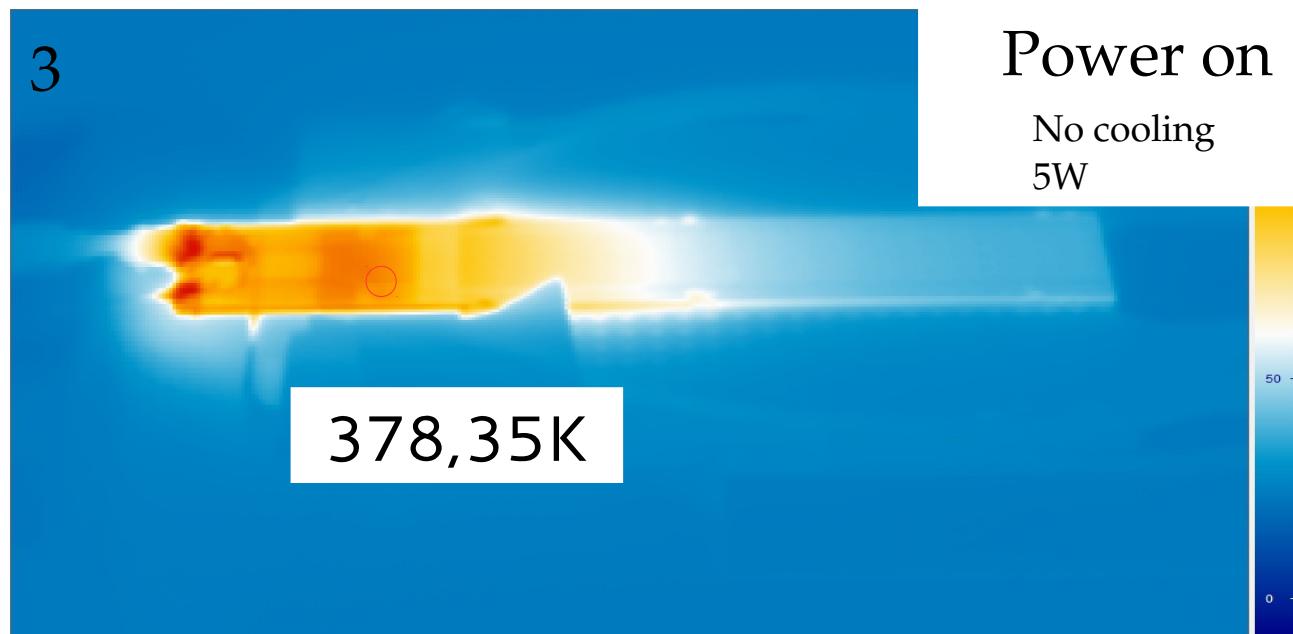
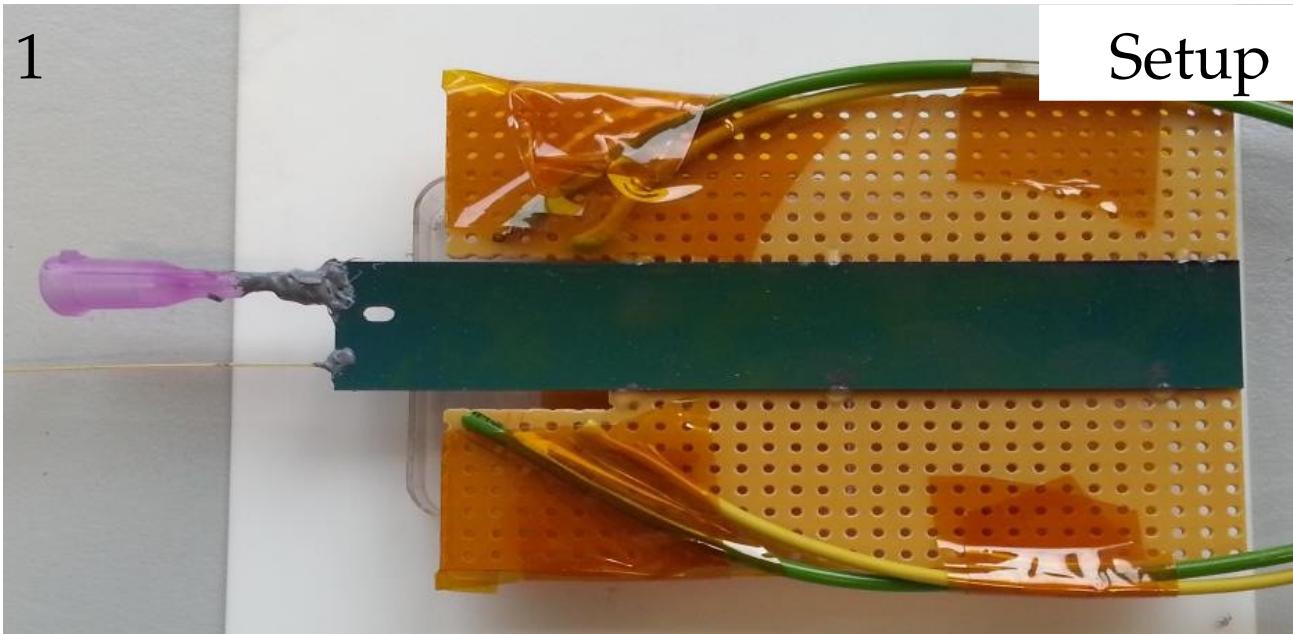
# Summary

- 1. Test vs simulation
  - 1.1 Test
  - 1.2 Simulation
  - 1.3 Comparison
- 2. CO<sub>2</sub> as an option?
- 3. Results with other coolants
  - 3.1 Coolants properties
  - 3.2 Coolants comparison
  - 3.3 Simulation results
- 4. Future work
- 5. Conclusions



# 1.1 Test with 5W and water as coolant

Test 1: 5W



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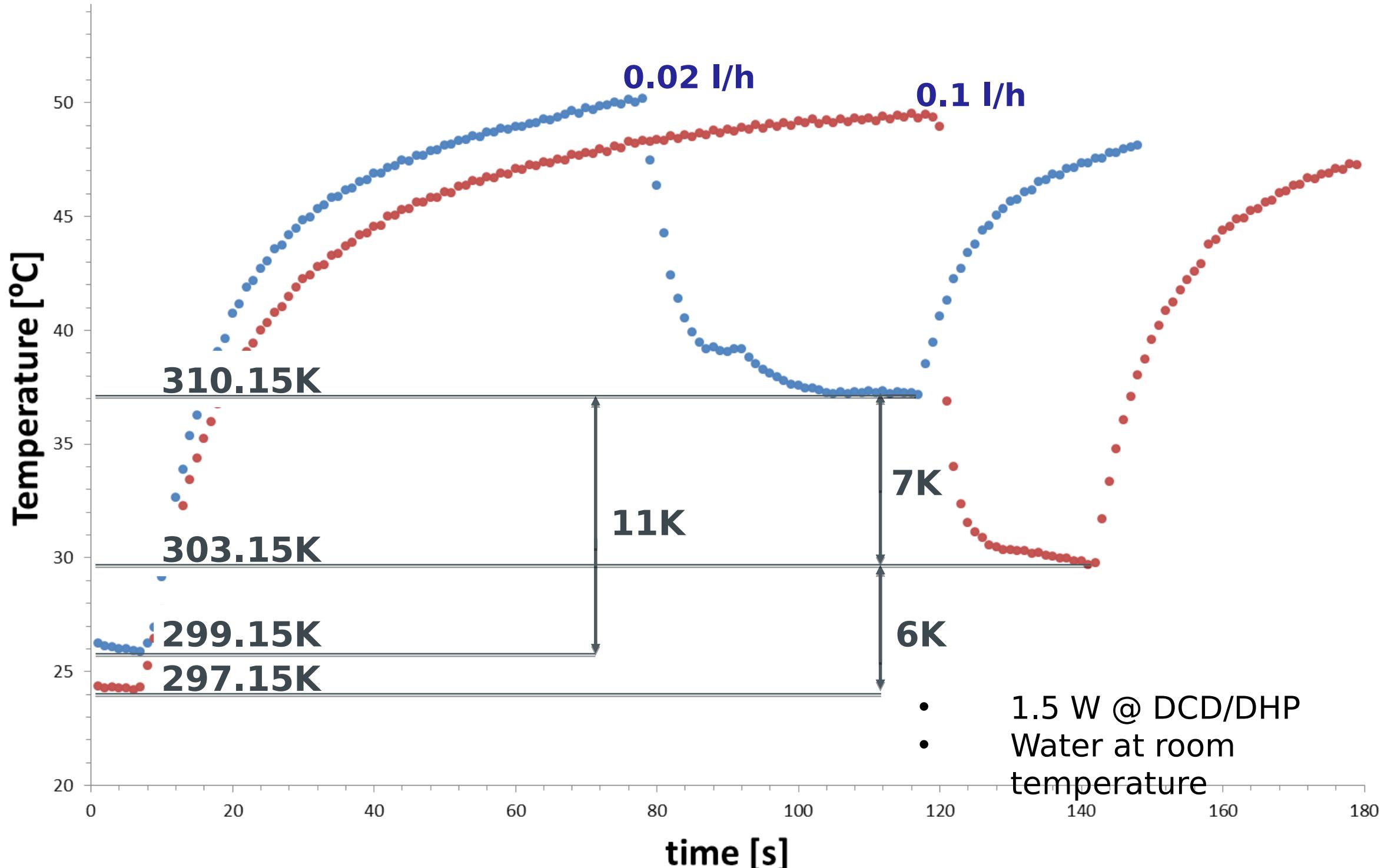
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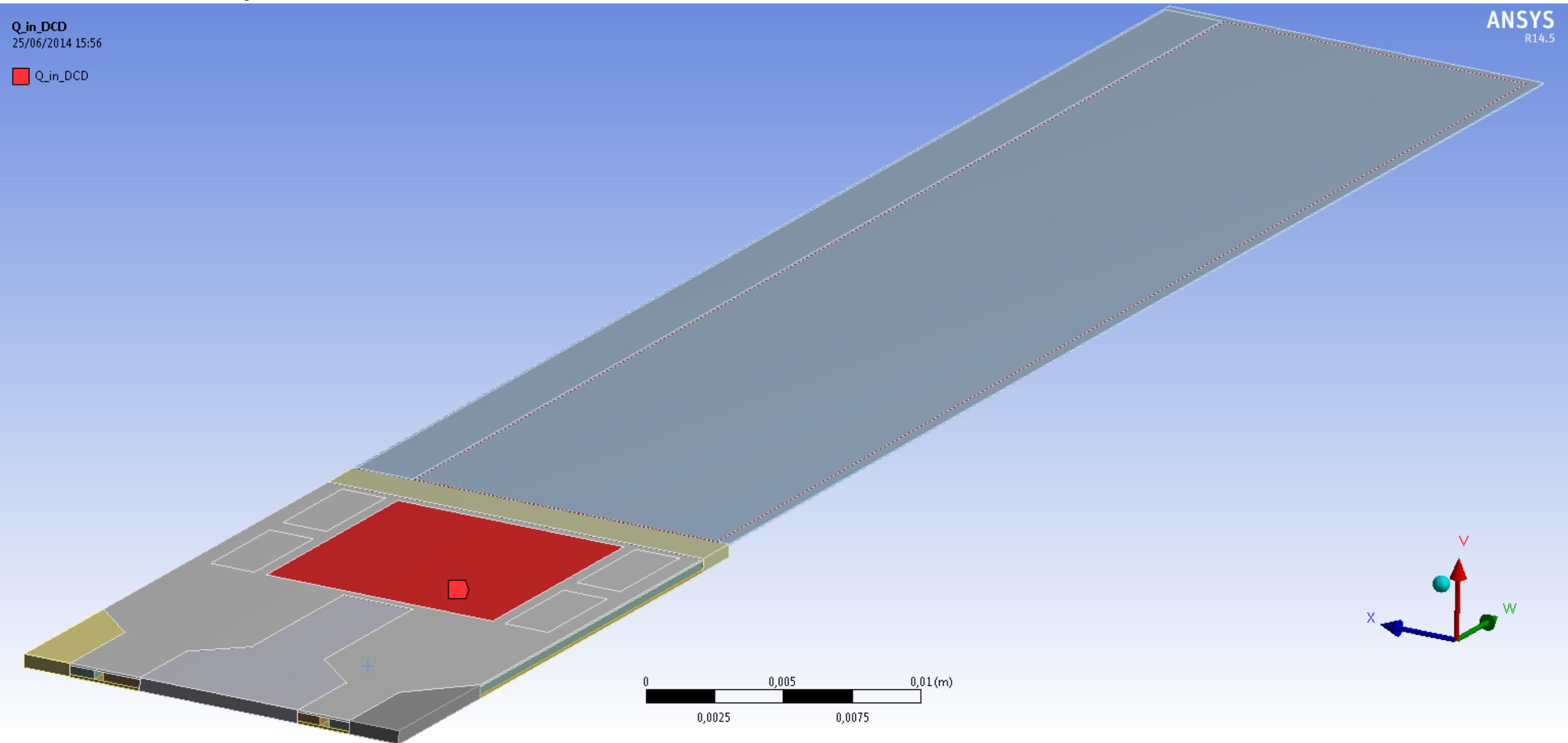
# 1.1 Test with 5W and water as coolant

Test 2 and 3: 1.5W with different volumetric flow



# 1.2 Simulations with 5W and water as coolant

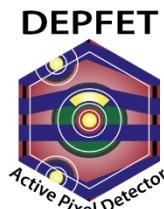
Boundary conditions:  $Q_{in}$



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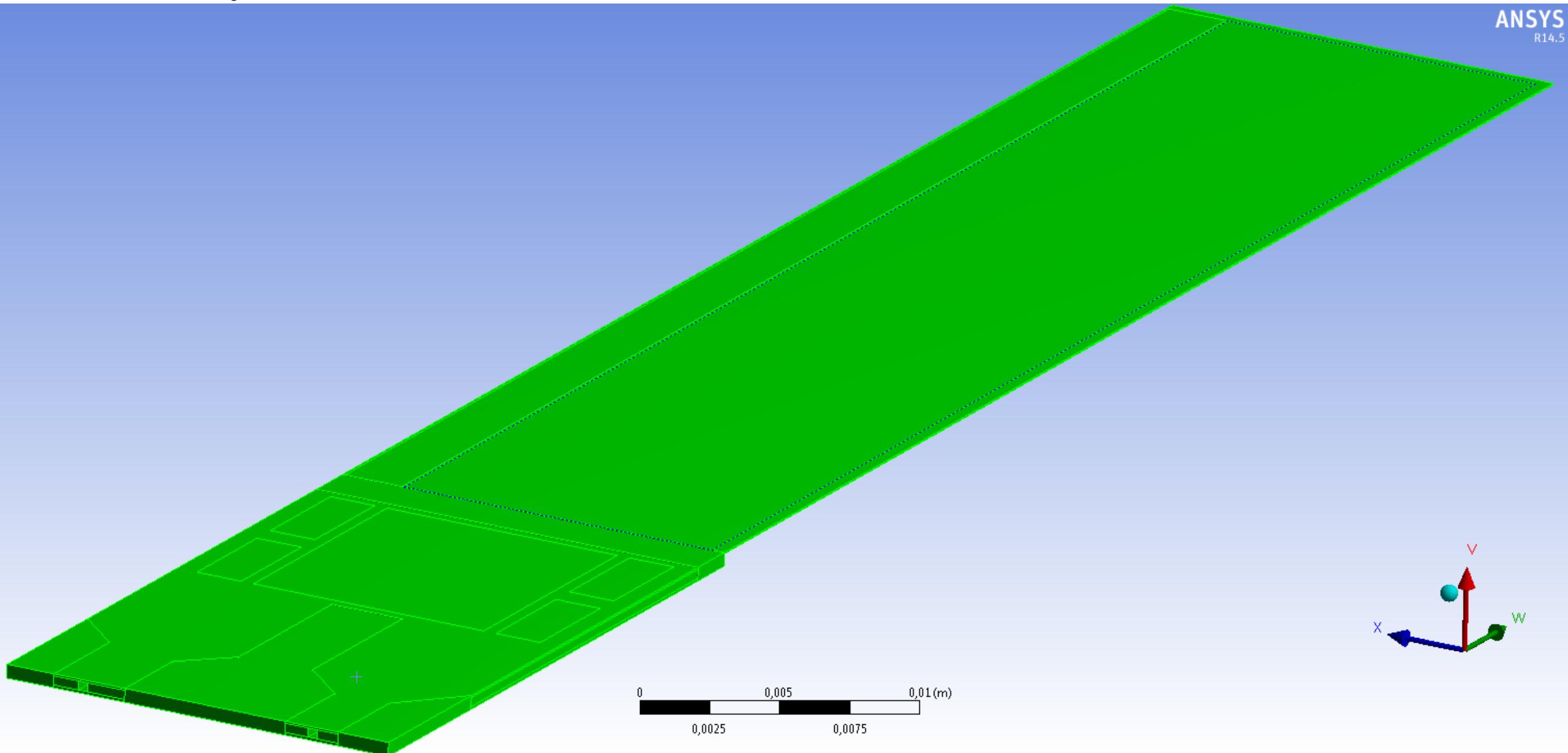
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# 1.2 Simulations with 5W and water as coolant

Boundary conditions:  $Q_{out}$



$T_{out} = \text{at } 297.15\text{K}, 299.15\text{K} \text{ and } 304.45\text{K}$   
 $h=5 \text{ [W m}^{-2} \text{ K}^{-1}\text{]}$



Update in the microchannel cooling simulations of  
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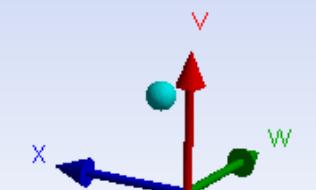
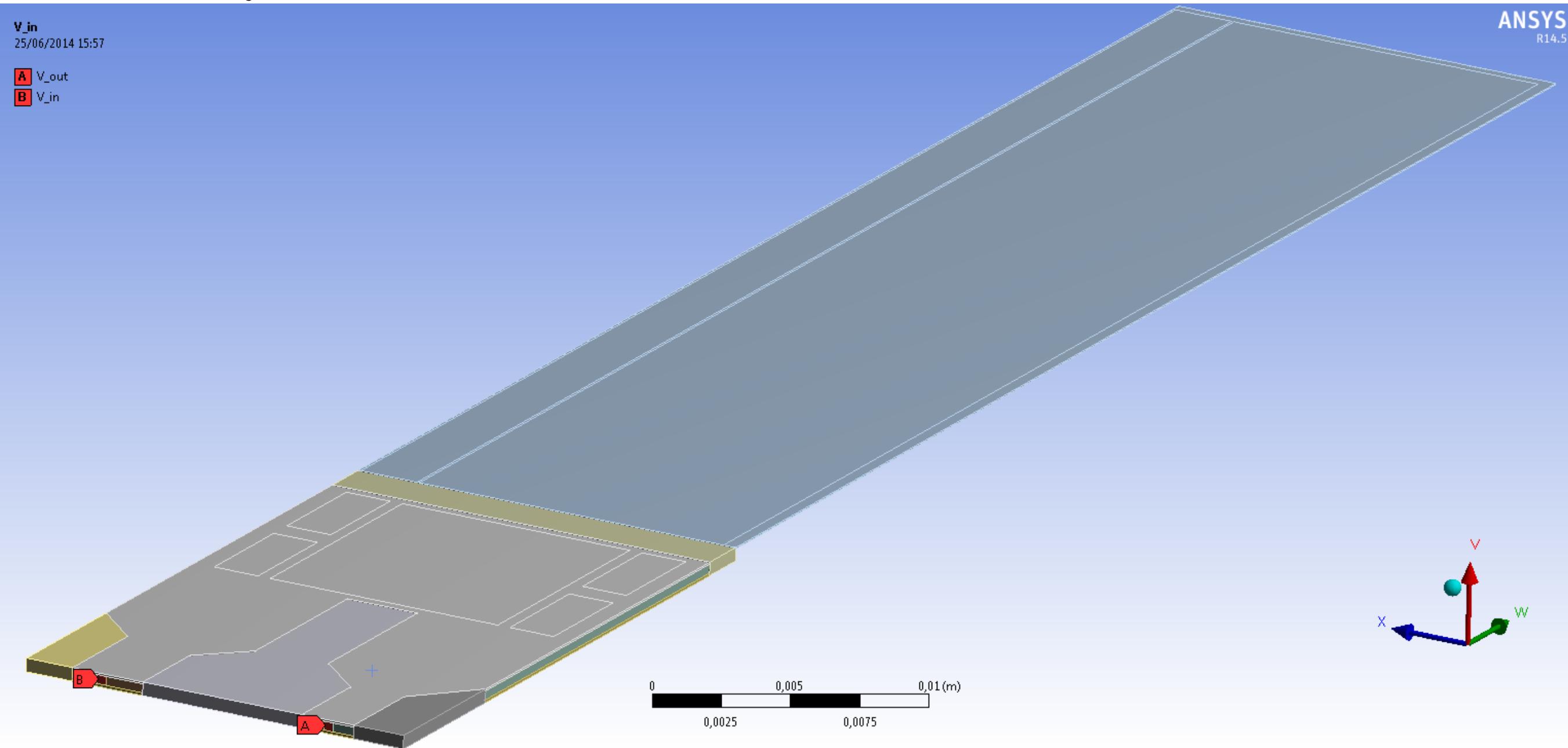
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# 1.2 Simulations with 5W and water as coolant

Boundary conditions: V\_in&V\_out



$V_{in}=2.15\text{m/s}$  (0.1 l/h) and  $0.43\text{m/s}$  (0.02 l/h)  
Fluid: water at 297.15K, 299.15K and 304.45K in the entrance



Update in the microchannel cooling simulations of  
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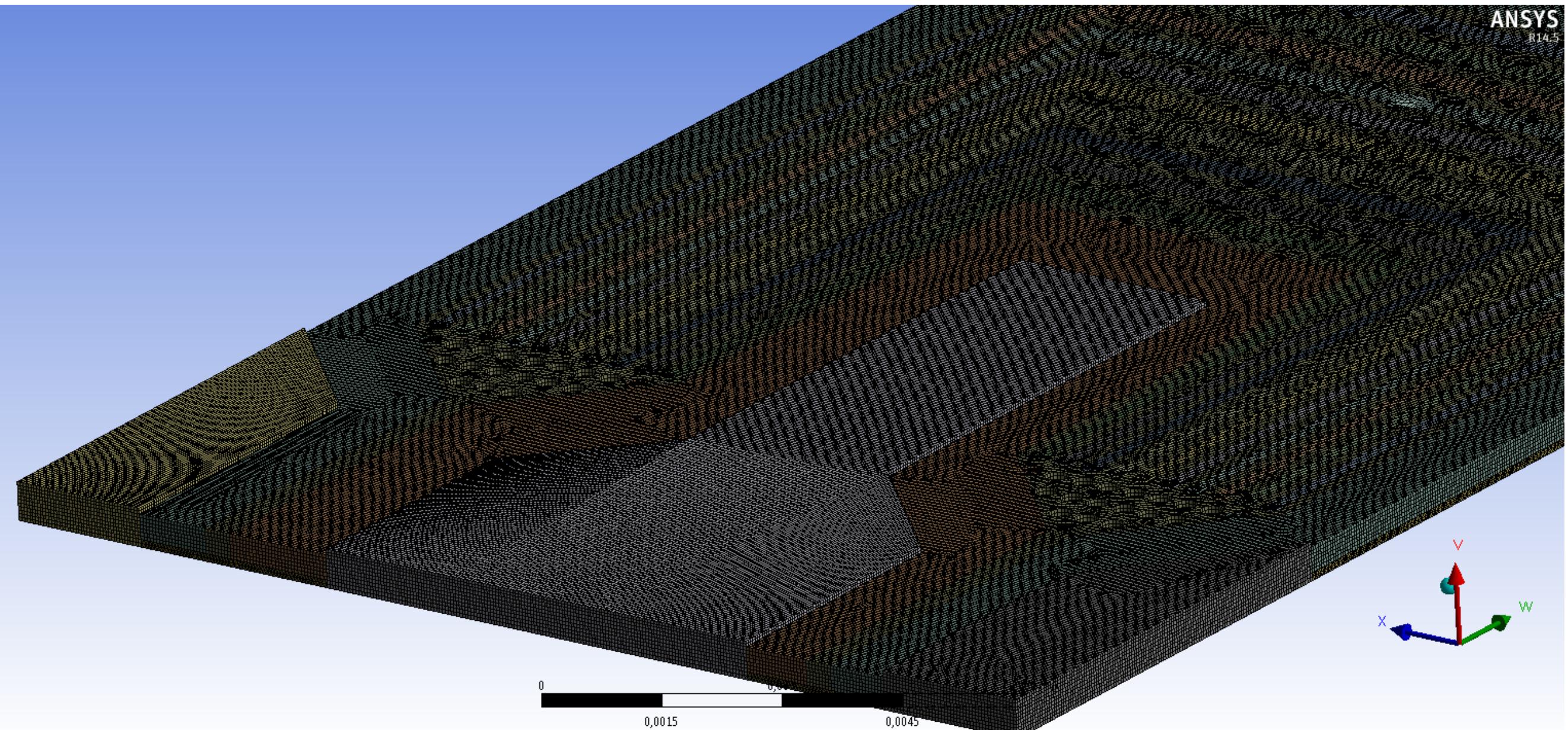
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# 1.2 Simulations with 5W and water as coolant

Mesh



5,15M elements divided mostly in 30% Hex8 and 70% Tet4



Update in the microchannel cooling simulations of  
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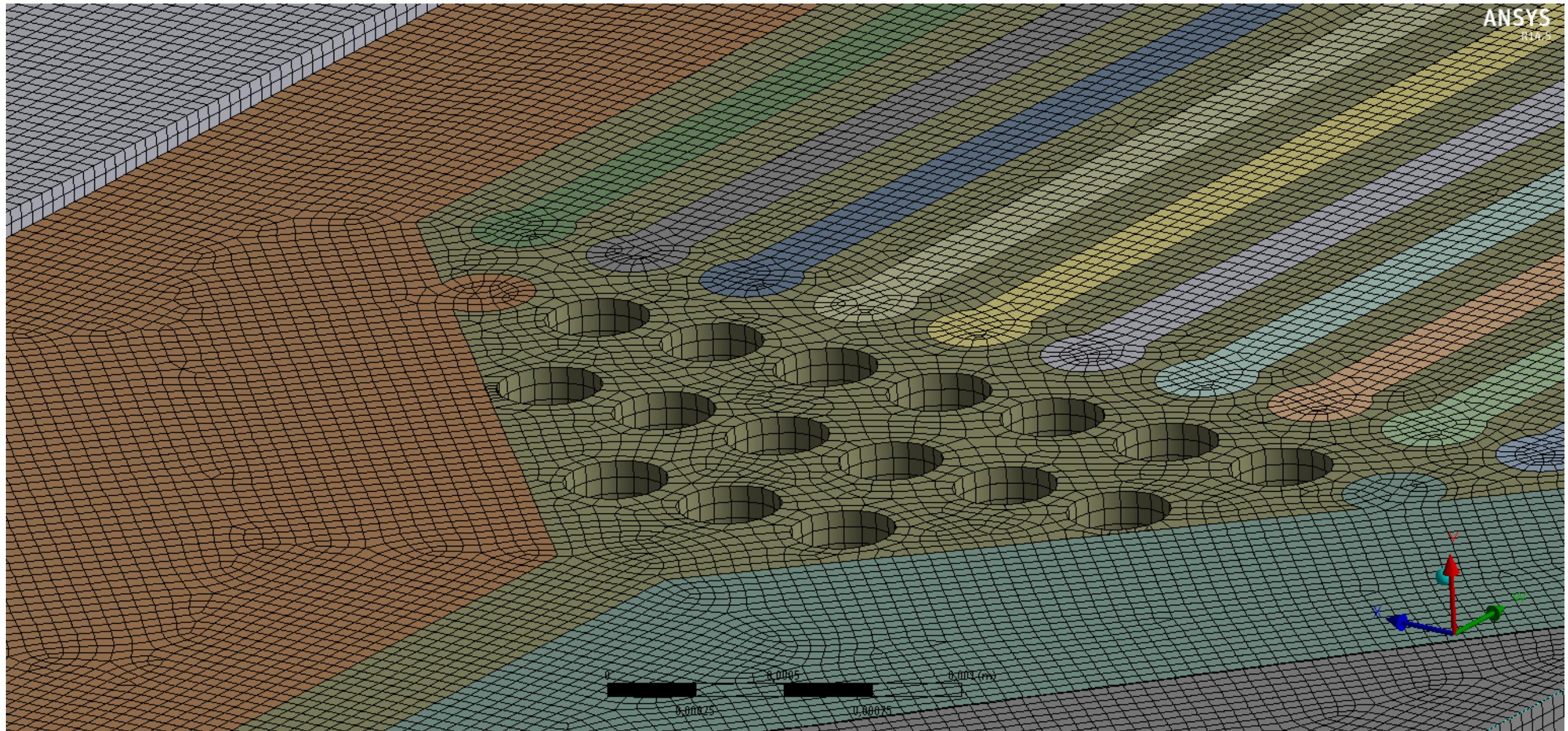
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# 1.2 Simulations with 5W and water as coolant

Mesh: detail



5,15M elements divided mostly in 30% Hex8 and 70% Tet4



Update in the microchannel cooling simulations of  
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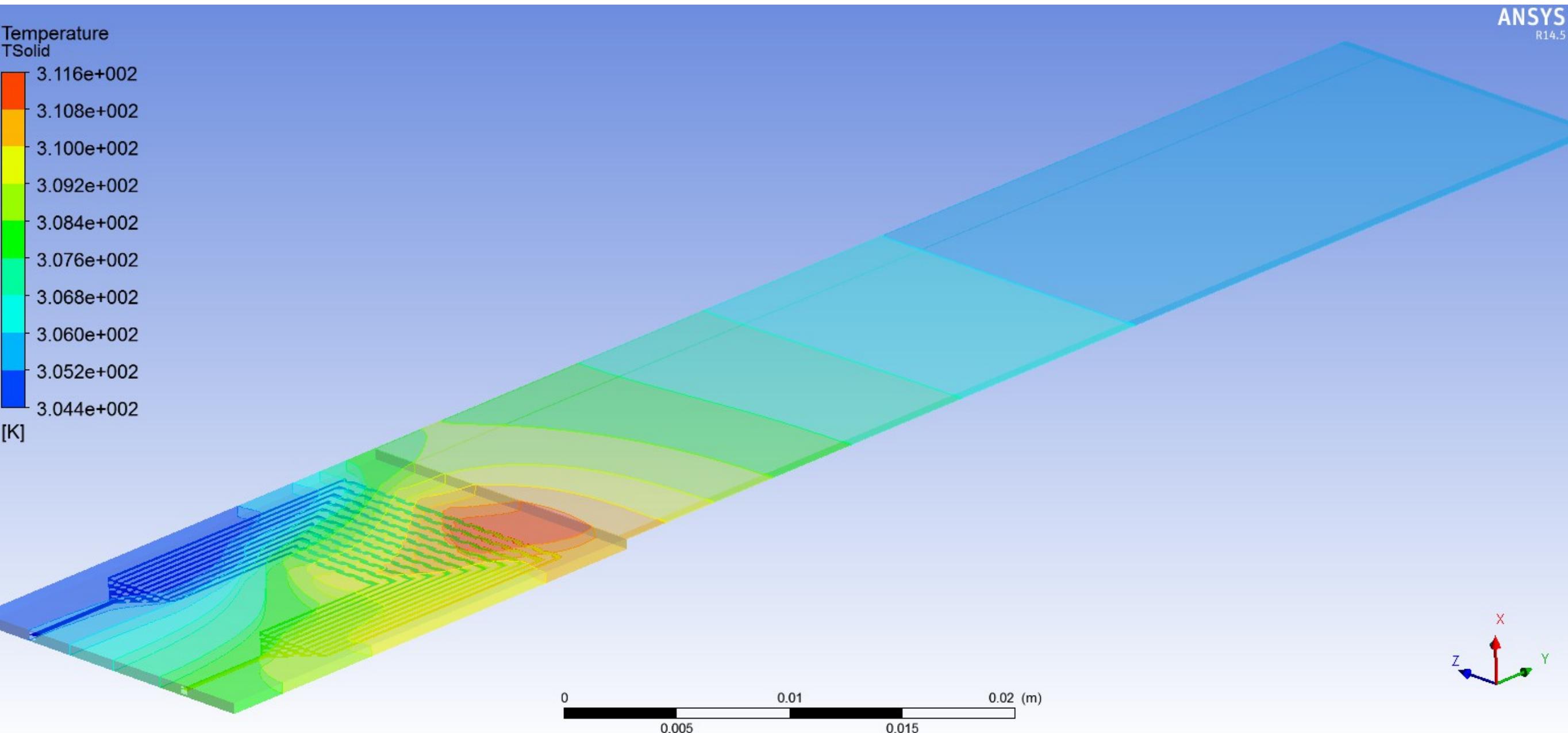
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# 1.2 Simulations with 5W and water as coolant

Results: simulation of the test 1 (slide 3)



$$T_{water\_out} = 308.6 \text{ K}$$

$$Q_{water} = C_p \cdot m \cdot \Delta T = 4186 \cdot 2.15 \cdot 0.00038 \cdot 0.00034 \cdot 997 \cdot (308.6 - 304.4) = 4.87 \text{ W}$$

97% of the 5W is absorbed by the cooling flow



Update in the microchannel cooling simulations of  
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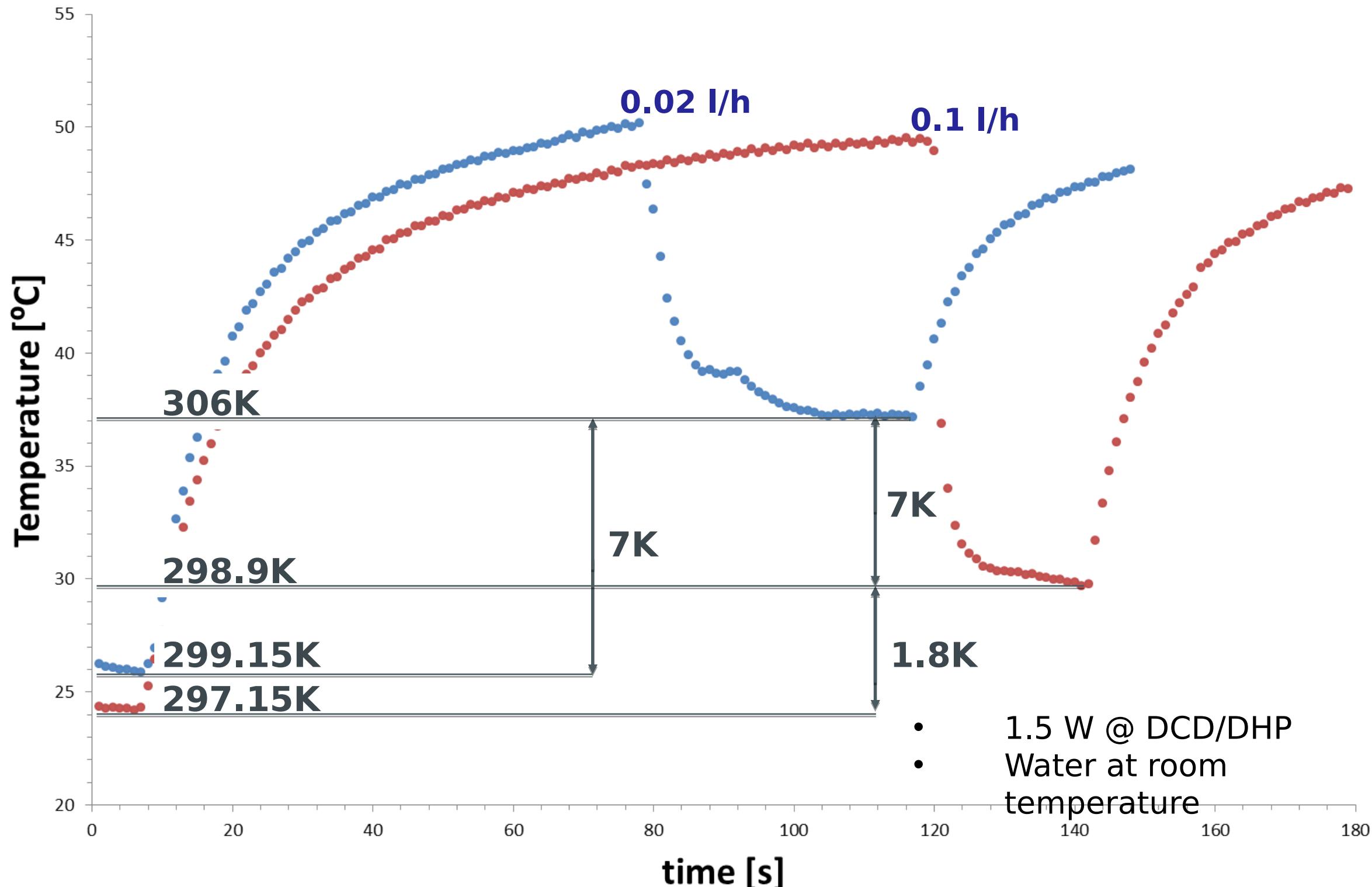
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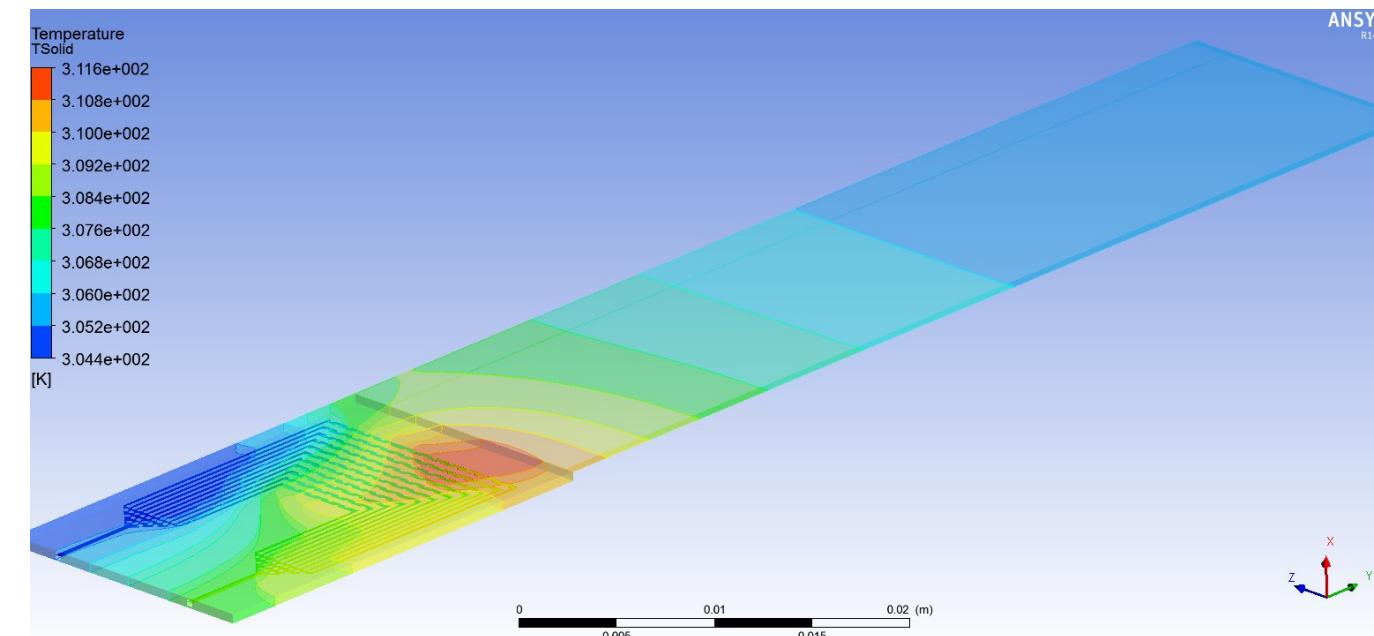
# 1.2 Simulations with 1.5W and water as coolant

Results: simulation of the test 2 and 3 (slide 4)



# 1.3 Comparison between tests and simulations

Test 1: 5W (0.11/h)



Simulation:  $\Delta T = 7.15\text{K}$

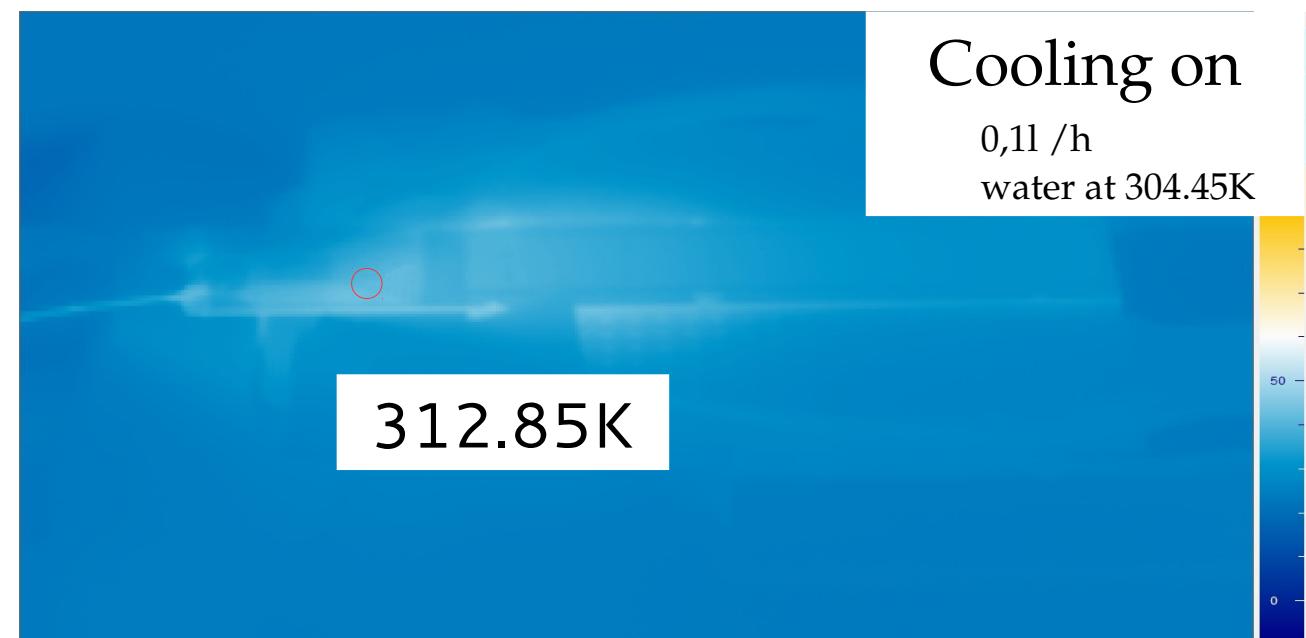
Test:  $\Delta T = 8.4\text{K}$



Initial state

Water and  
DEPFET at room  
temperature

304.45K



Cooling on

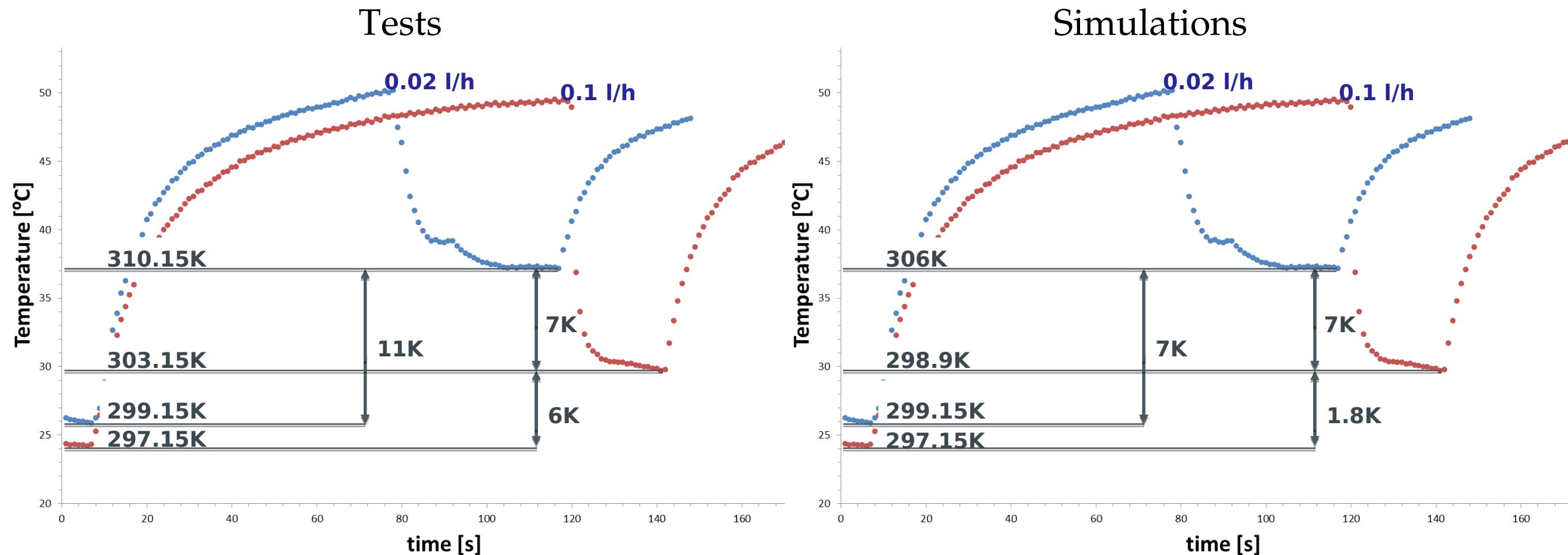
0,11 /h  
water at 304.45K

312.85K



# 1.3 Comparison between tests and simulations

Test 2 and 3: 1.5W with different volumetric flow

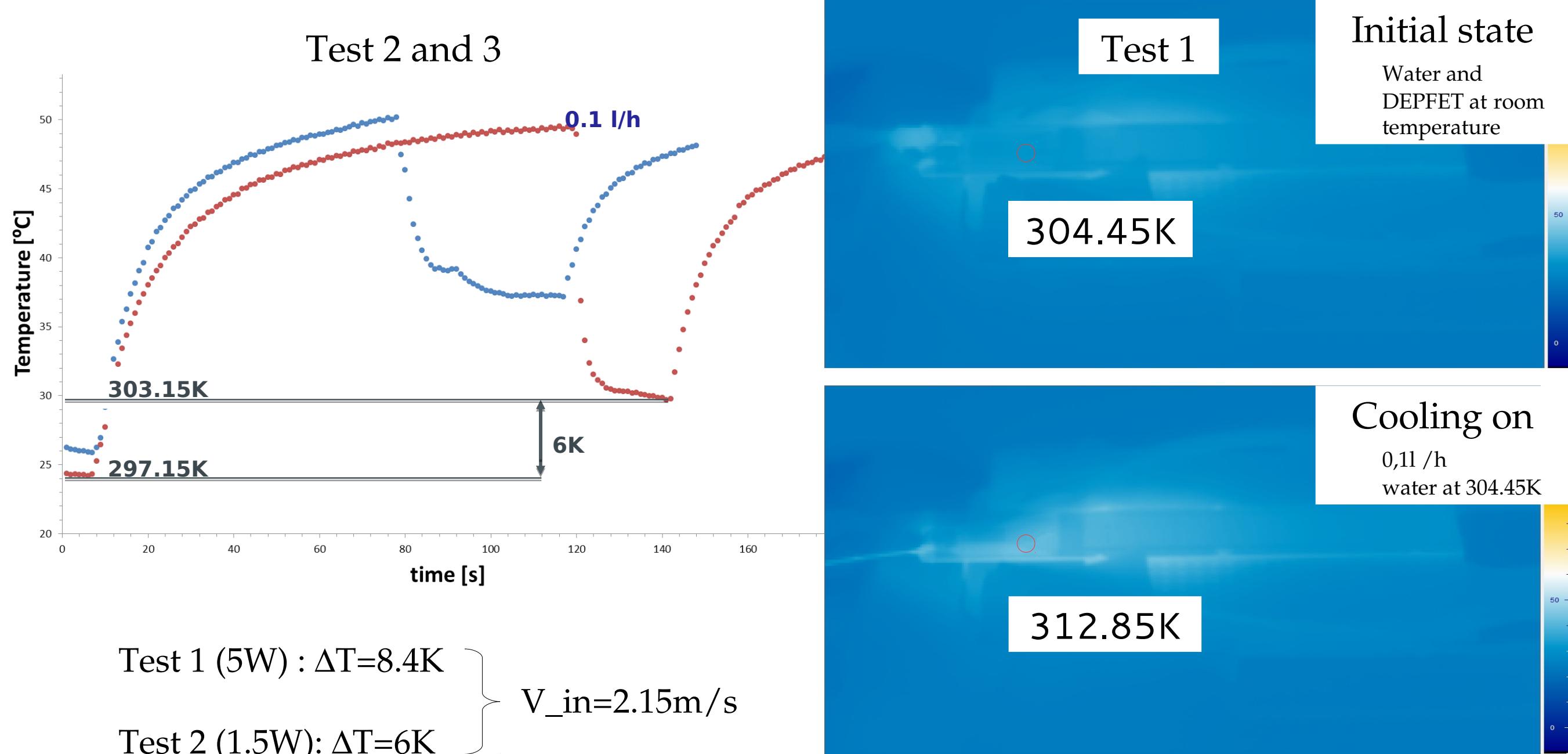


- Initial conditions: the same (297.15K and 299.15K)
- Final temperature:  $\Delta$  of 4K between tests and simulation (310.15K-306K and 303.15K - 298.9K)
- $\Delta T=7K$  with different flows in tests and simulations



# 1.3 Comparison between tests and simulations

Test 1, 2 and 3: 5W vs 1.5W



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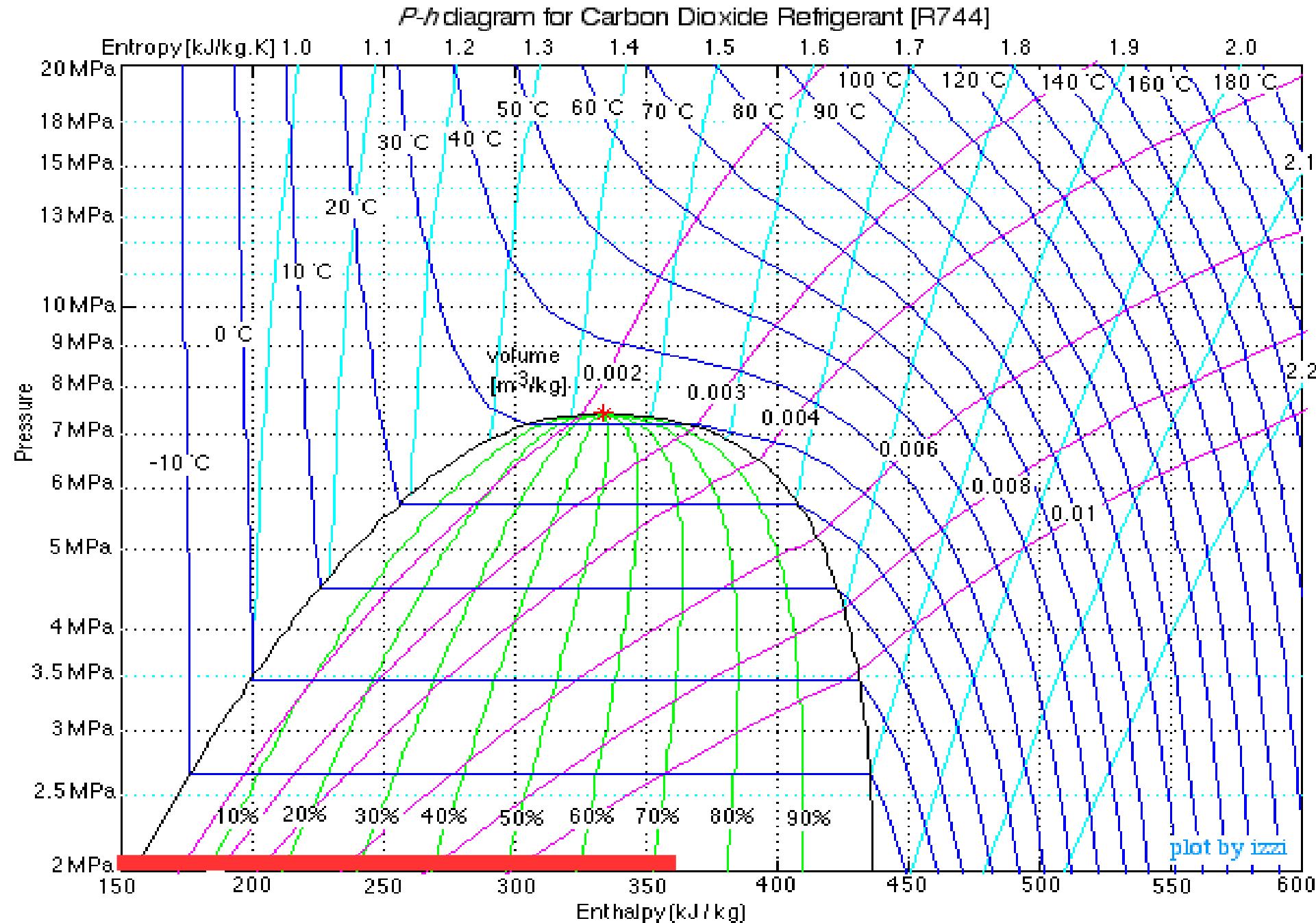
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## 2. CO<sub>2</sub> cooling

LHCb detectors op. Temperature ~ -40°C

ATLAS IBL detectors op. Temperature ~ -20°C



WHY CO<sub>2</sub>?

Low T → Low  $\mu$  → Low  $\Delta P$

Low frozen T

Compared mainly with fluorocarbon fluids at -40-20°C

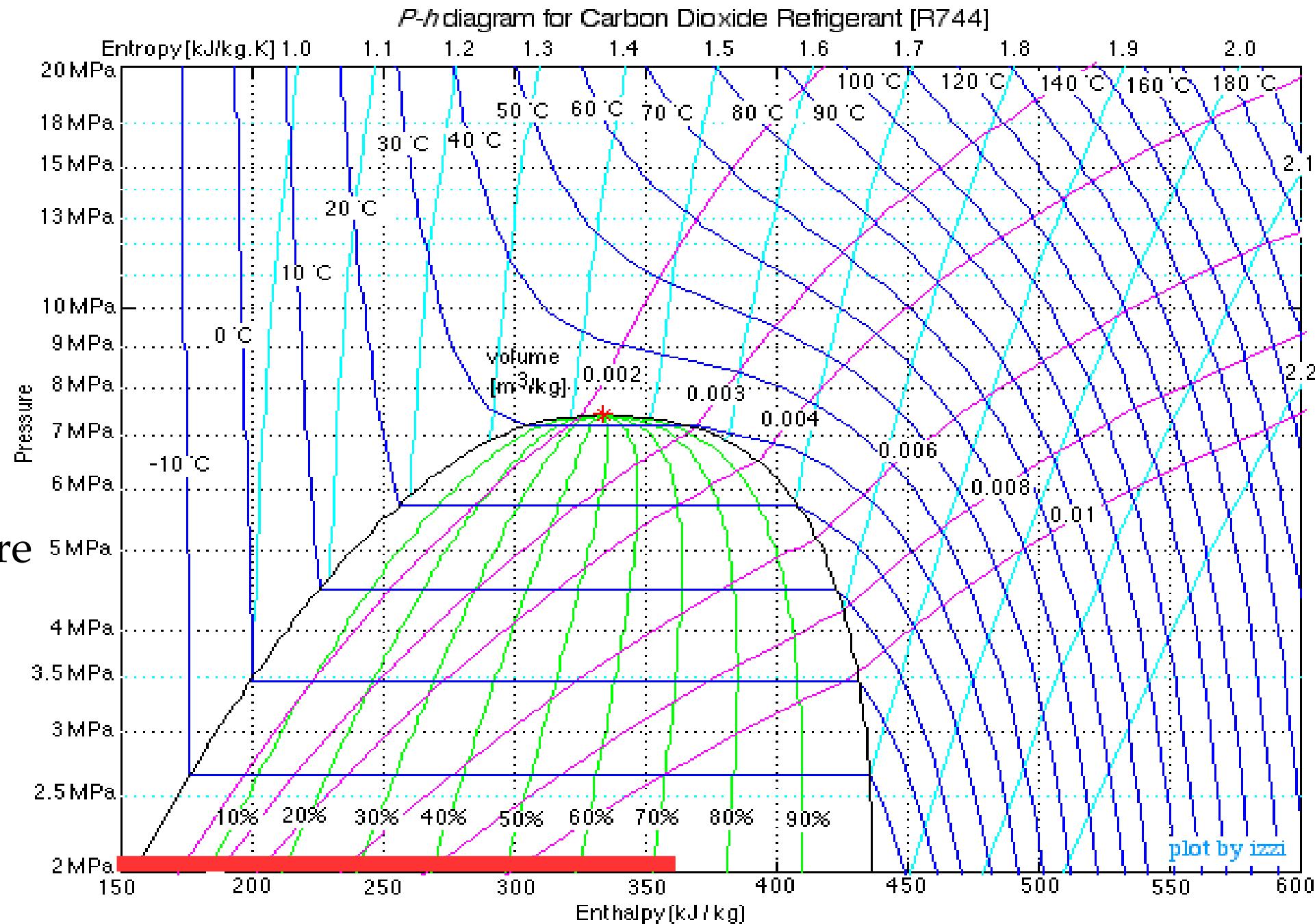


## 2. CO<sub>2</sub> cooling

DEPFET detectors op. Temperature  $\sim +25^\circ\text{C}$

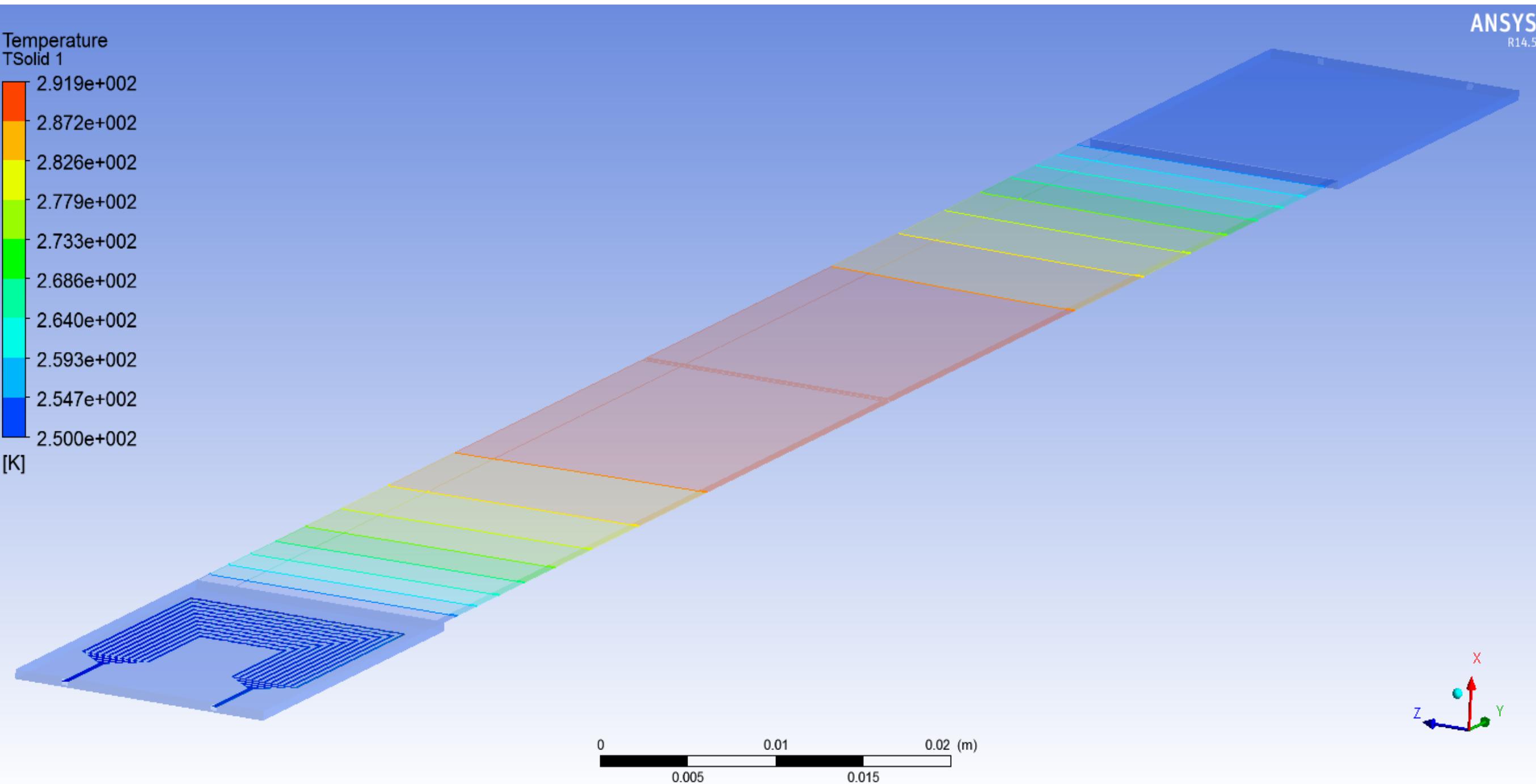
If P=20 bar (2MPa)

- Biphase fluid
- High  $\Delta T$  though the sensor (250-293K)
- Op. Fluid Temperature very different from the Op. DEPFET Temperature



## 2. CO<sub>2</sub> cooling

DEPFET detectors op. Temperature  $\sim +25^\circ\text{C}$

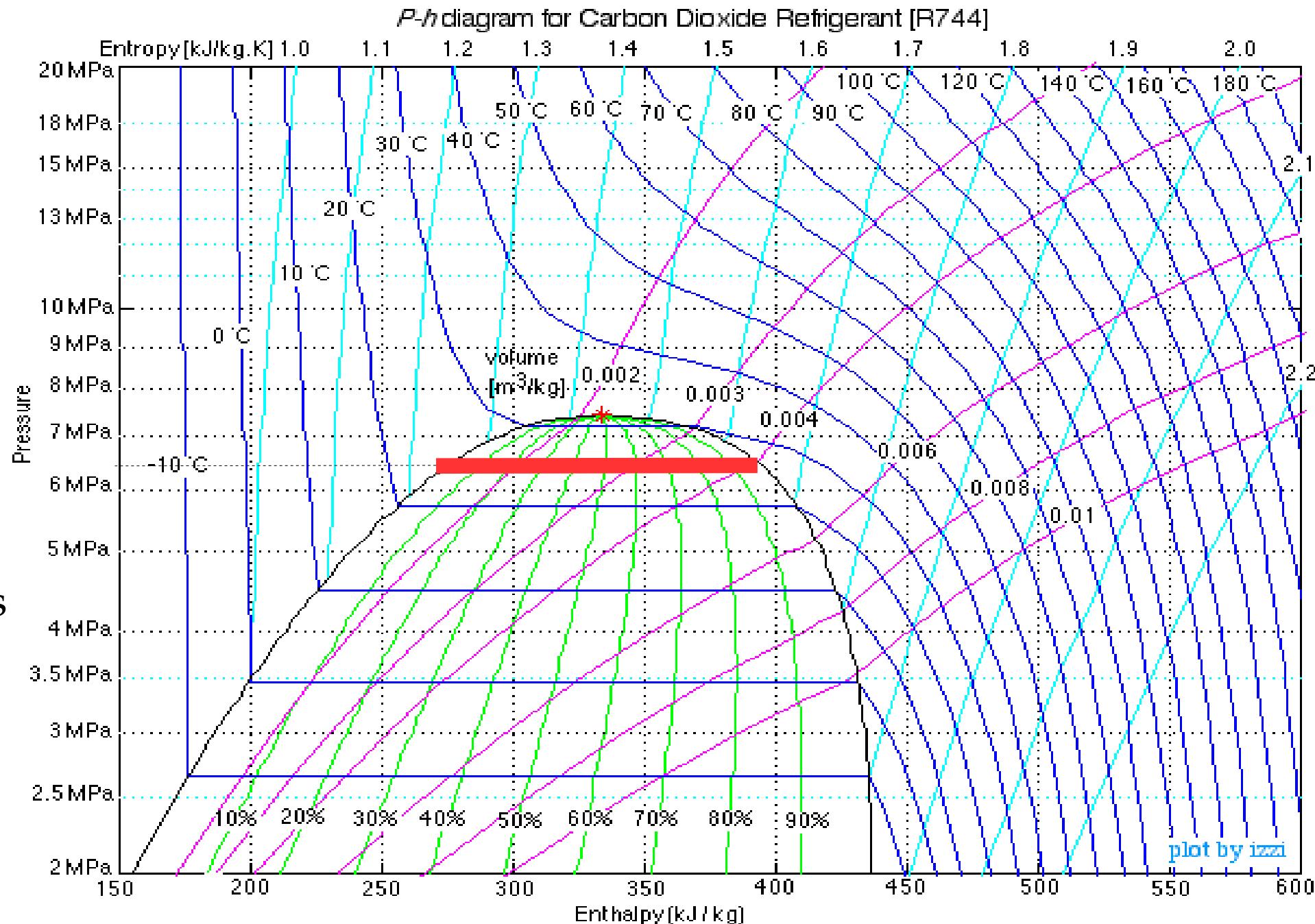


## 2. CO<sub>2</sub> cooling

DEPFET detectors op. Temperature  $\sim +25^\circ\text{C}$

If P=65 bar (6.5MPa)

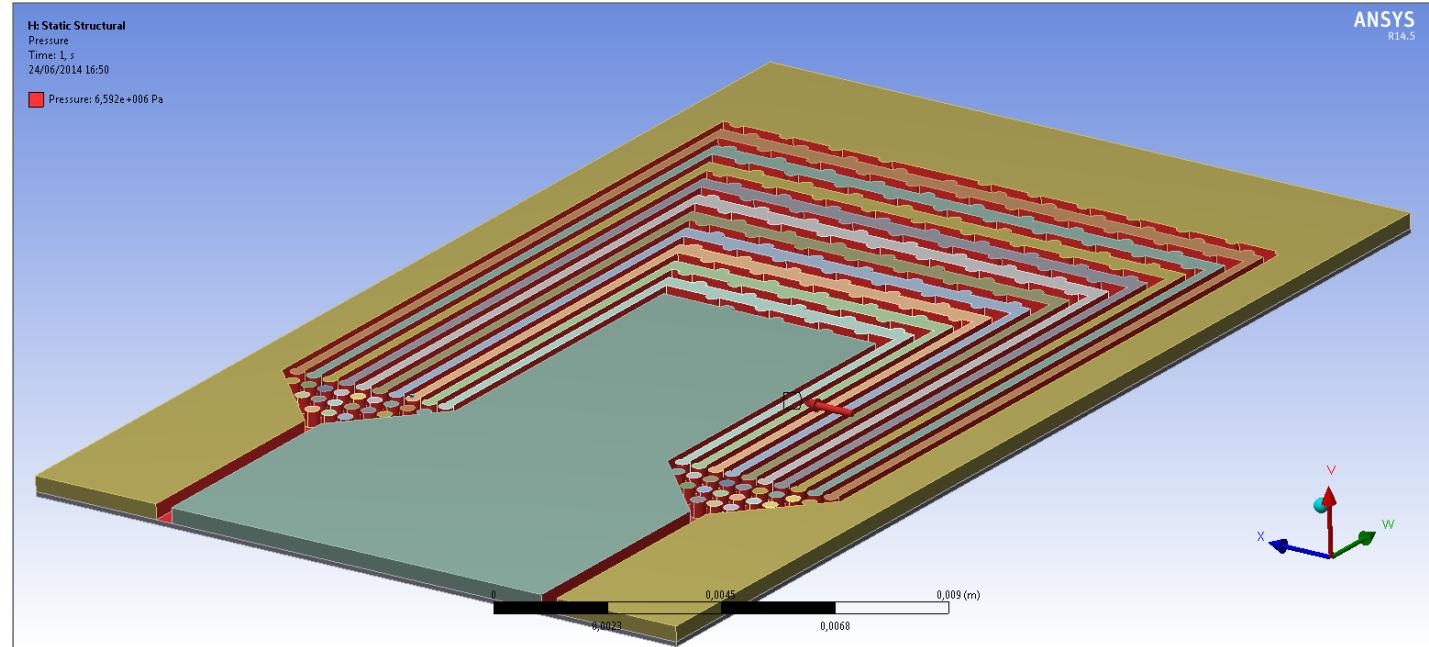
- Expensive
- Dangerous
- Pure water is better option in heat transfer ( $C_p, \lambda$ ).
- Extra mechanical stress in the microchannels
- Biphase fluid



## 2. CO<sub>2</sub> cooling

DEPFET detectors op. Temperature  $\sim +25^\circ\text{C}$

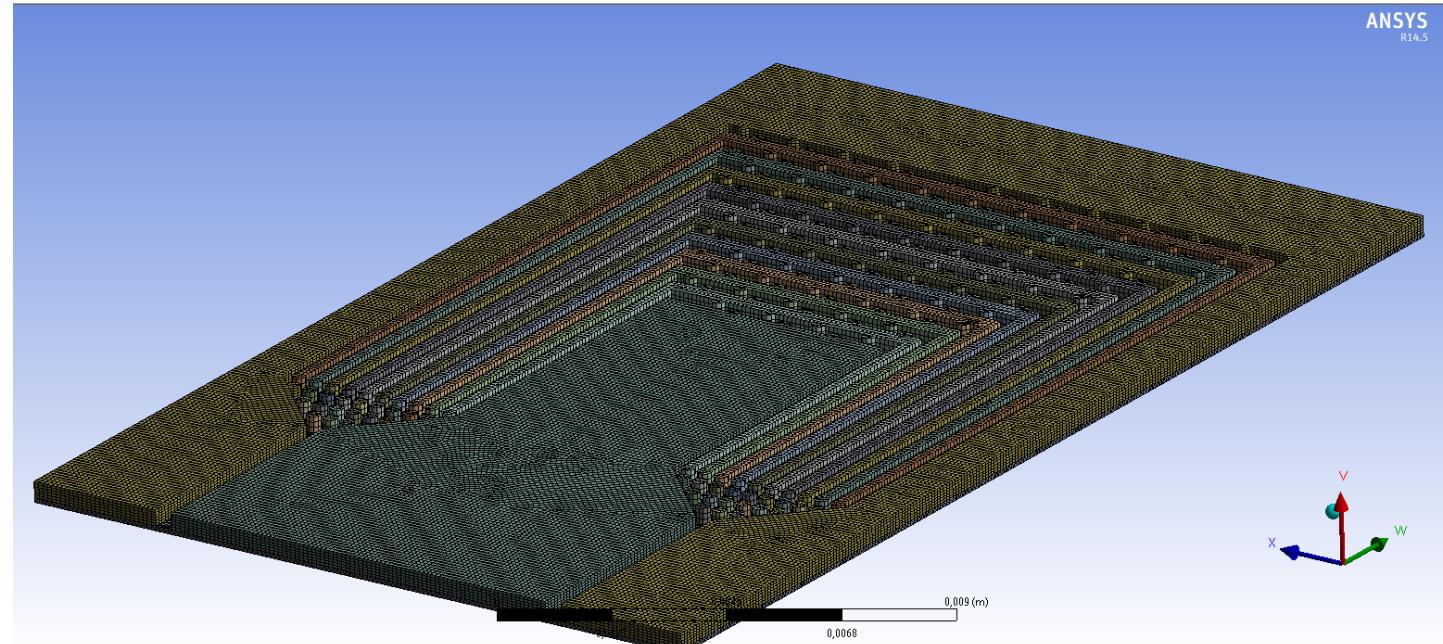
If P=65 bar (6MPa)



About 600K elements

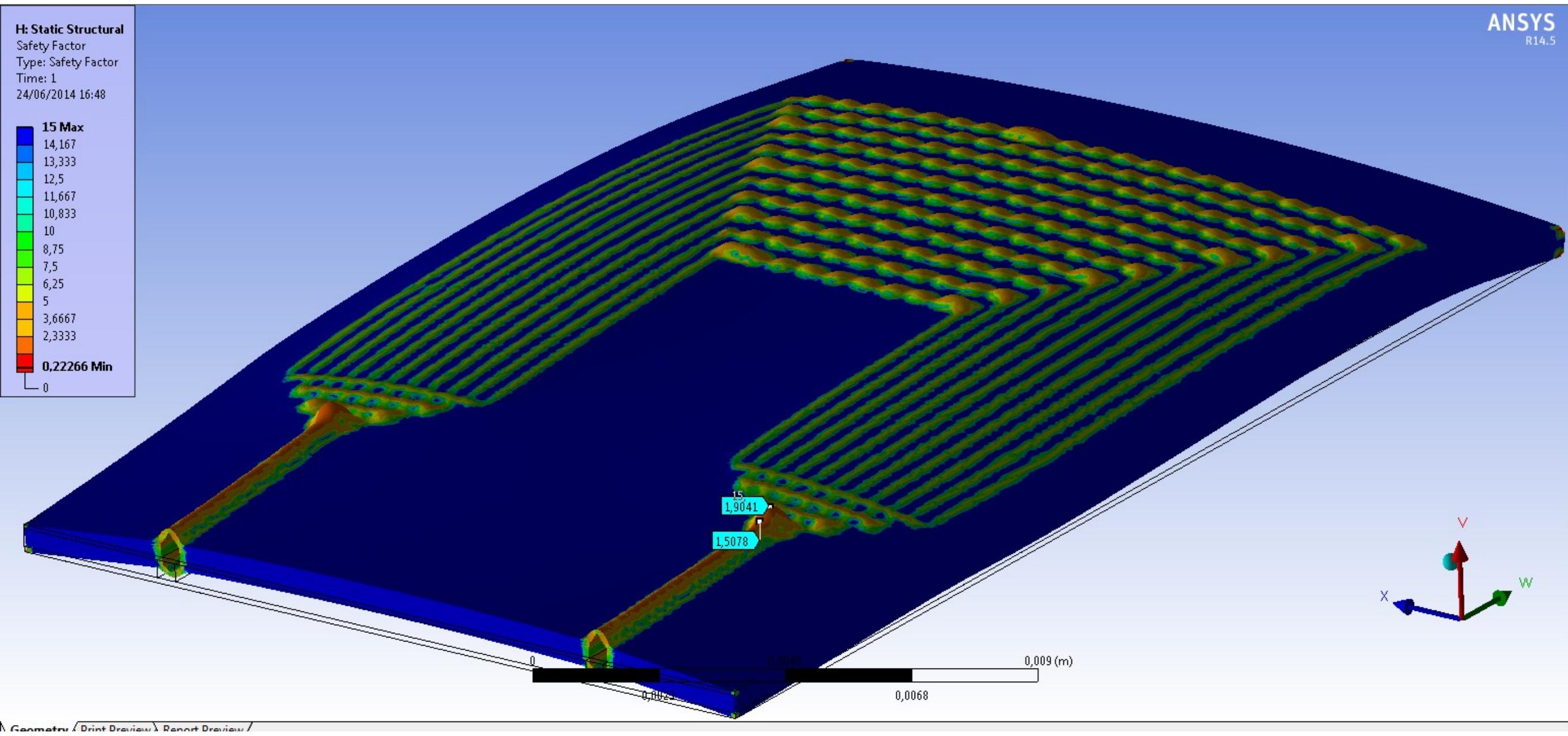
75% Tet10

25% Hex20



## 2. CO<sub>2</sub> cooling

DEPFET detectors op. Temperature ~ +25°C



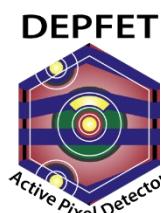
SF=Max. Stress allowed by the material / Simulated stress



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# 3.1 Simulations with other coolants: main properties

Fluid $(21\text{-}25^\circ\text{C})$ 1 atm	Pure Water	Air	PGW2080	PGW6040	EGW2080	EGW6040
$C_p [\text{J kg}^{-1} \text{K}^{-1}]^1$	4181.7	1004.4	4077.164	3537.17	3809.26	3047.408
$\lambda [\text{W m}^{-1} \text{K}^{-1}]^1$	0.6069	2.61E-02	0.49305	0.32524	0.49824	0.34946
$\rho [\text{kg m}^{-3}]^1$	997.0	1.185	1018.56	1047.52	1034.88	1098.56
$\mu [\text{kg m}^{-1} \text{s}^{-1}]^1$	8.899E-04	1.831E-05	1.95E-03	9.51E-03	1.61E-03	5.17E-03
Relative permittivity <sup>2</sup>	80.2	1.00059	PG pure: 32		EG pure: 37.7	
$X_0 [\text{mm}]^3$	360.7	309367.09	364.4	371.8	357.7	351.6

PGW2080: 20% Propylene Glycol, 80% Pure Water

PGW6040: 60% Propylene Glycol, 40% Pure Water

EGW2080: 20% Ethylene Glycol, 80% Pure Water

EGW2080: 60% Ethylene Glycol, 40% Pure Water

<sup>1</sup>Manufacturer technical data sheet

<sup>2</sup>[www.matweb.com](http://www.matweb.com)

<sup>3</sup>own calculation



## 3.2 Coolants Comparison: PGW vs EGW

### Advantages PGW

- Less dense
- High Cp
- No toxic
- Radiation length
- Low relative electric permittivity
- Environmenta friendly

### Advantages EGW

- Less viscous
- High thermal transfer coefficient  $\lambda$
- More stable with temperature differences
- Better understood (more history)



## 3.2 Coolants comparison: Water vs PGW&EGW

### Advantages Designized Water

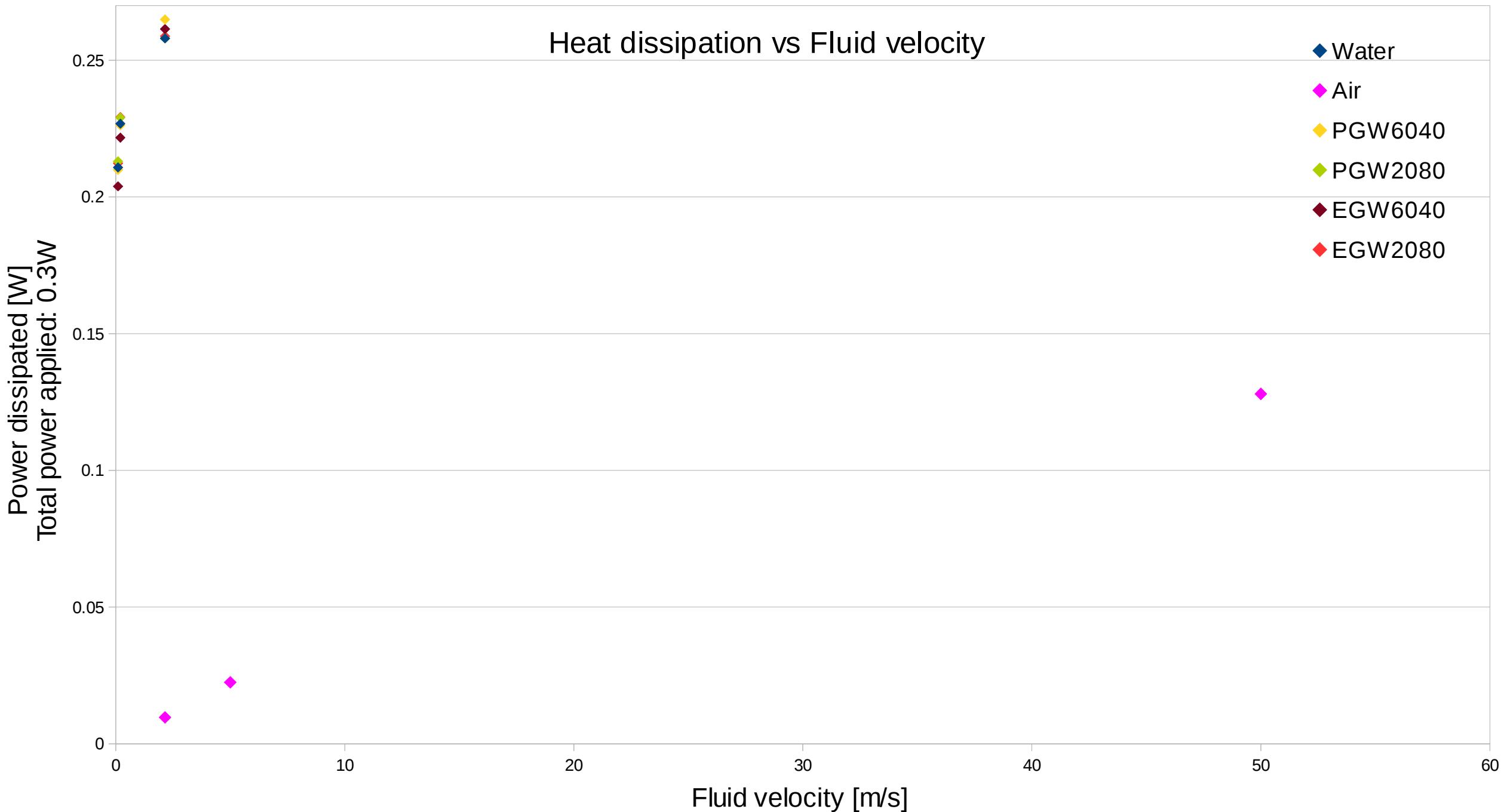
- Cheaper
- Higher Cp
- Higher thermal transfer coefficient  $\lambda$
- Less viscous  $\mu$

### Advantages PGW&EGW

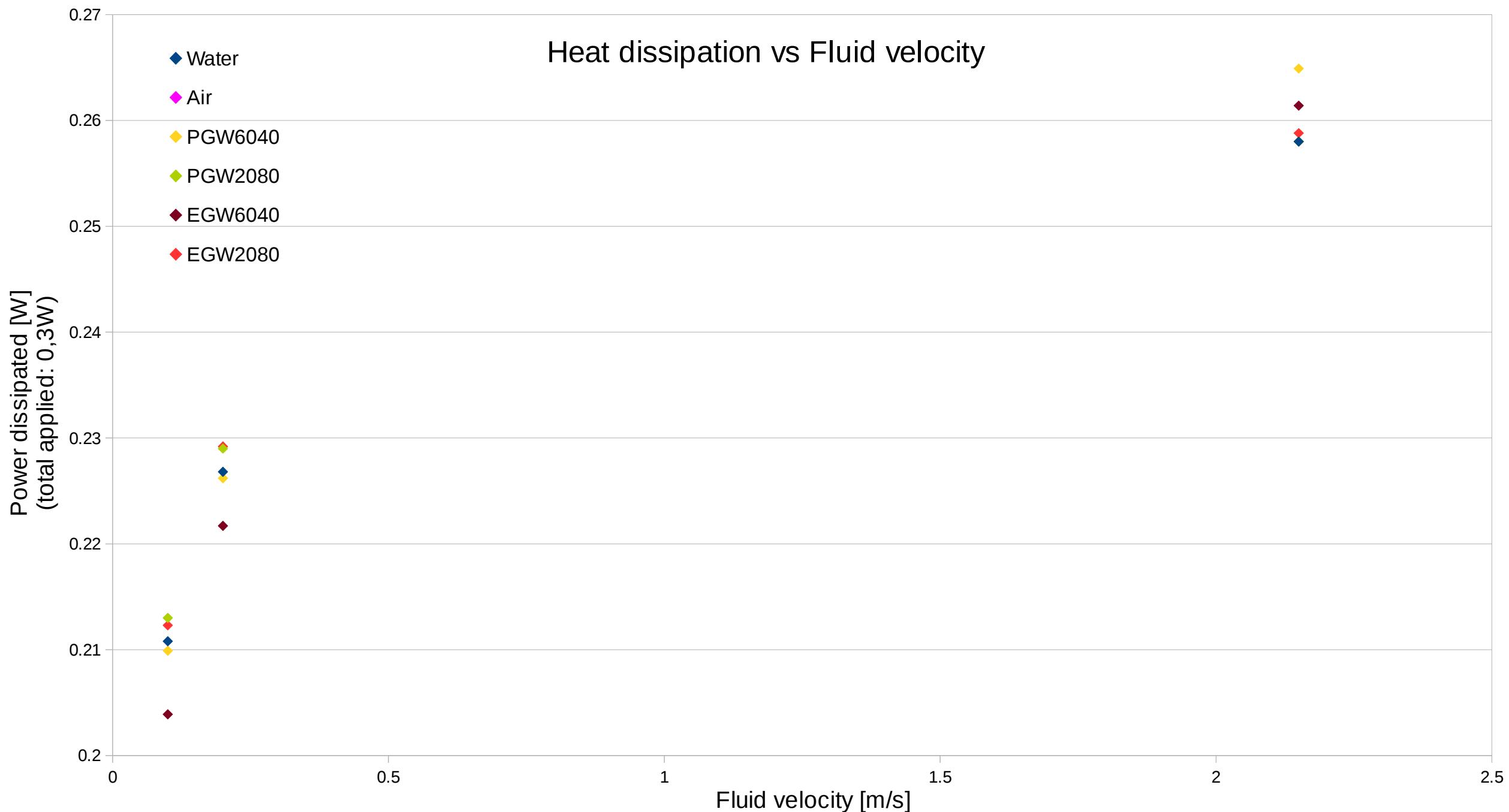
- Less corrosive
- Low relative electric permittivity
- Higher boiling point and lower freezing point
- Higher Vapor Pressure (easier to evaporate)
- Less Denser



### 3.3 Coolants simulation result



### 3.3 Coolants simulation result



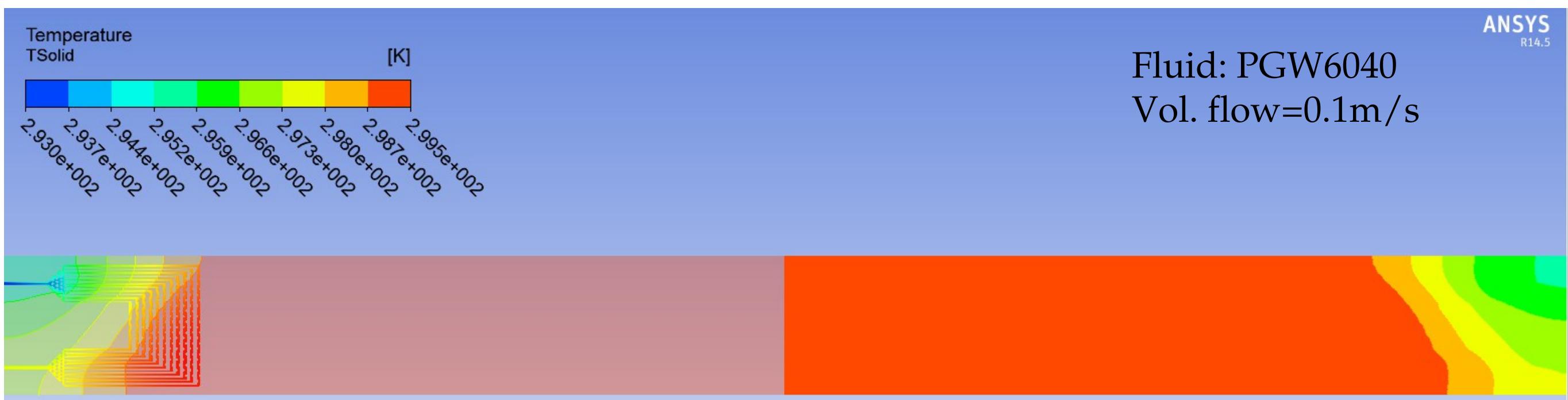
### 3.3 Coolants simulation result with current geometry

High vol. flow (2.15m/s)

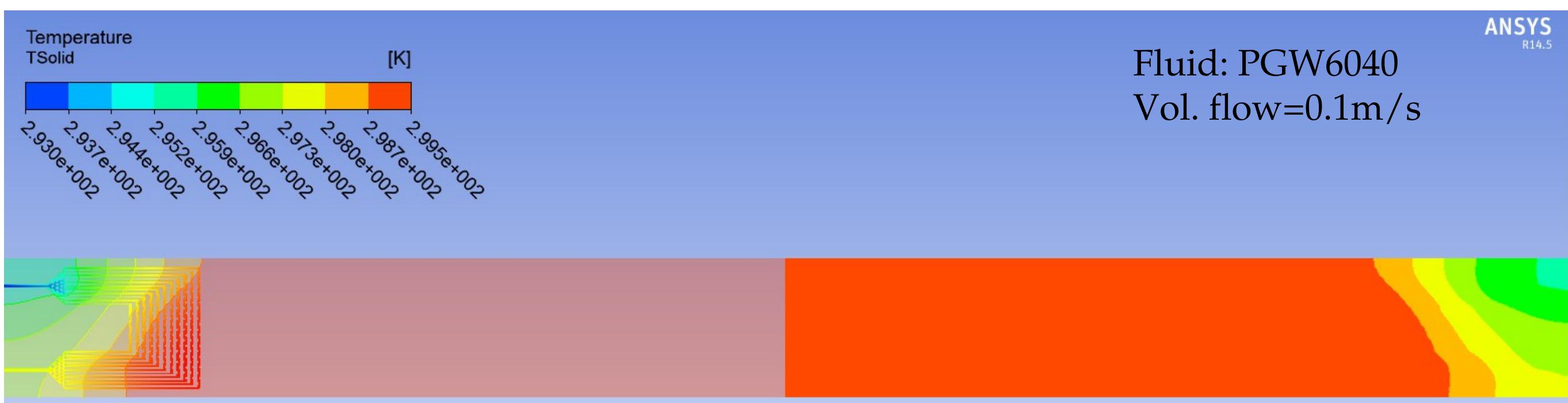
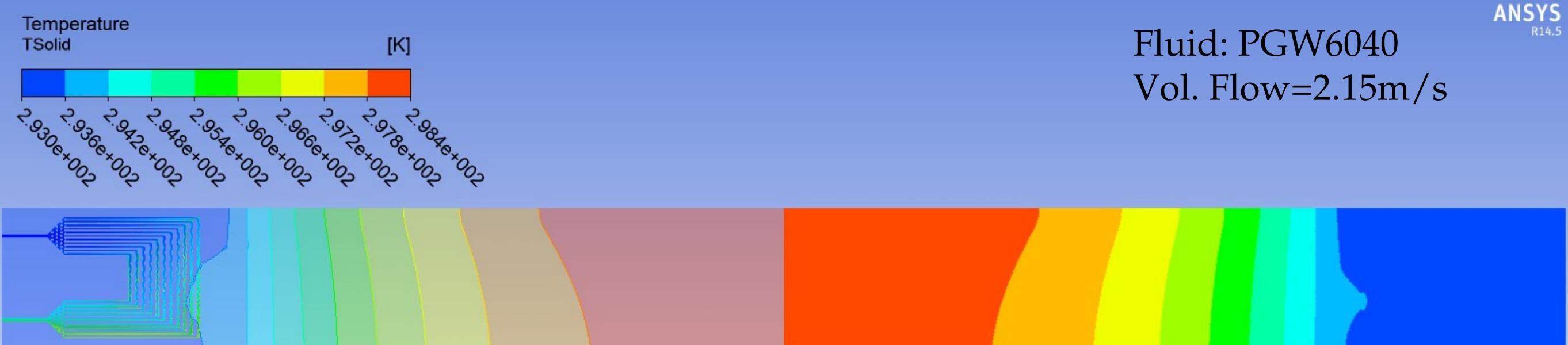
- More heat dissipated
- High  $\Delta T$  through the sensor
- High cross section
- High material budget

Low vol. flow (0.1-0.2m/s)

- Less heat dissipated
- Homogeneous T through the sensor
- Possibility to reduce cross section increasing the mass flow
- Less cross section less material



### 3.3 Coolants simulation result with current geometry



### 3.3 Coolants simulation result with current geometry

High vol. flow (2.15m/s)

- More heat dissipated
- High  $\Delta T$  through the sensor
- High cross section
- High material budget

Low vol. flow (0.1-0.2m/s)

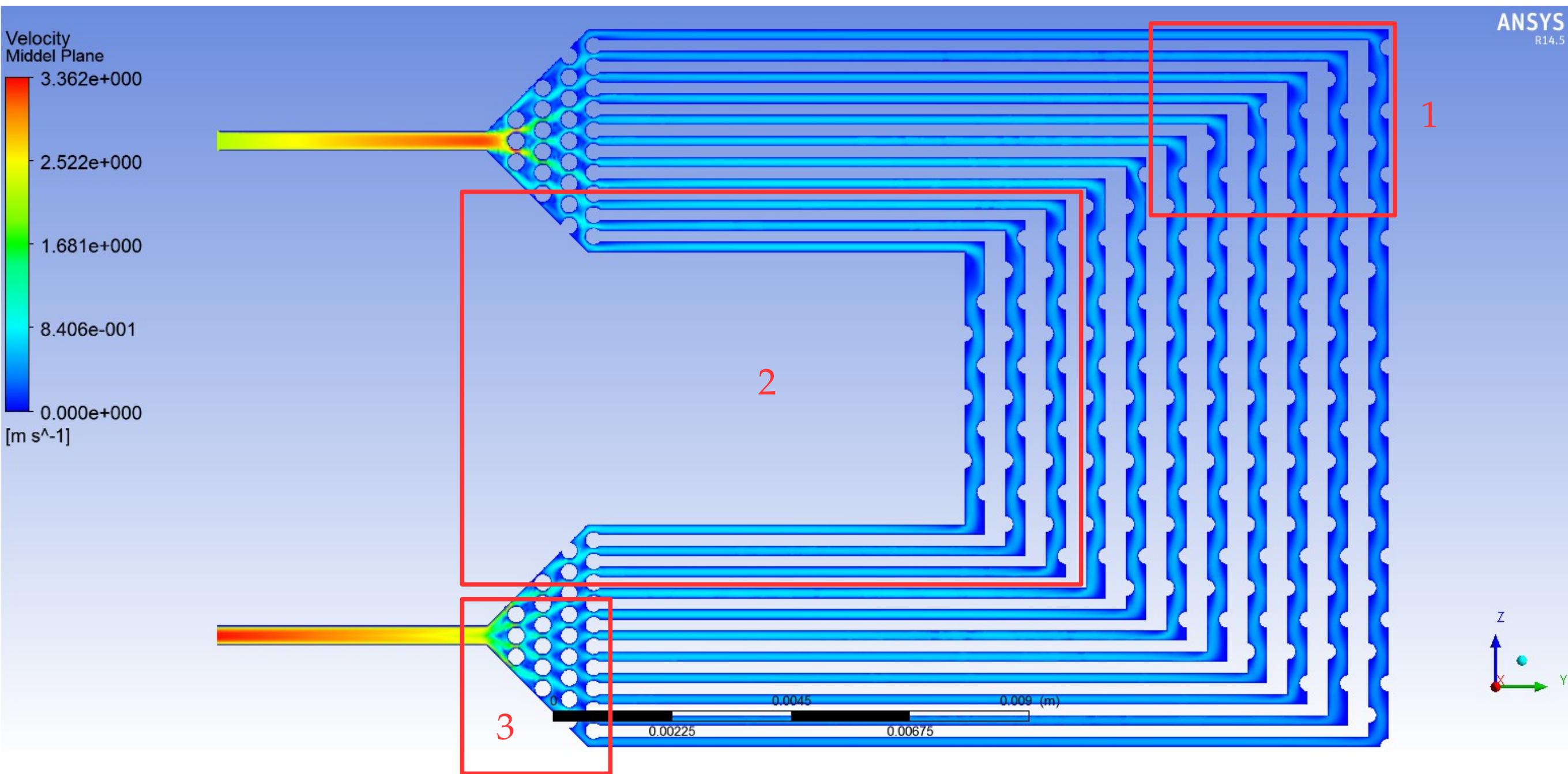
- Less heat dissipated
- Homogeneous T through the sensor
- Possibility to reduce cross section increasing the mass flow
- Less cross section less material

Proposed solution:

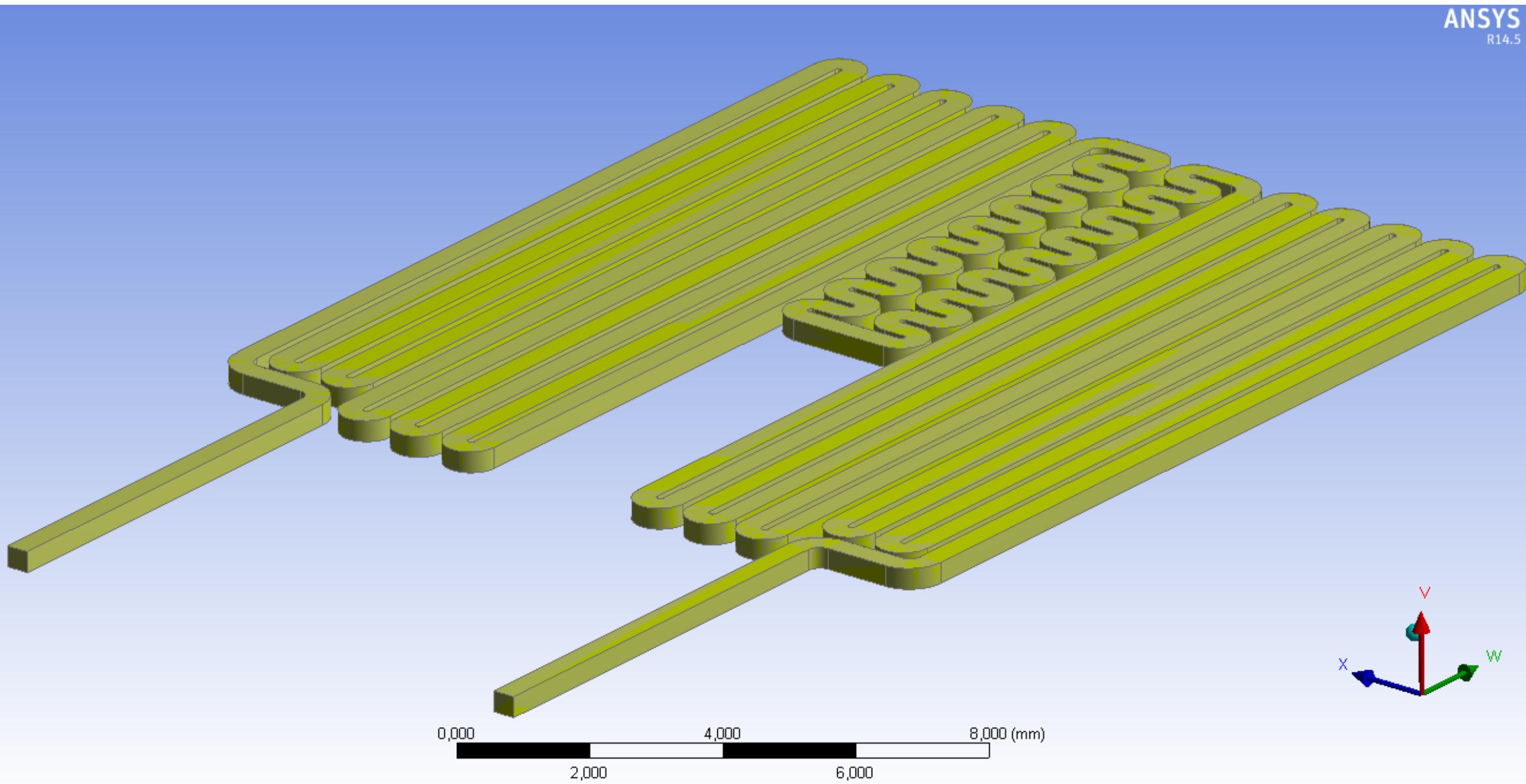
V: 0.15m/s  $\Delta T$  near to 0 → we can save material (Si and fluid if we decrease the cross section) but to achieve the DT through the sensor near to 0 → rise up the volumetric flow to 2,15m/s then usage of PGW6040 which has better thermal performance



# 4 Microchannel alternative geometry



# 4 Microchannel alternative geometry



# 5. Future work

- Try to proximate (even more) simulations with test
- Material properties dependence with temperature (Si, water,...)
- Refine mesh
- Thermal stress and deformations
- Multiphase simulations (if needed)
- Simulations with more detailed elements (volume in switches, interface layer between chips and detector, ...)
- Optimize the microchannels geometry



# 6. Conclusions

- Simulations match with tests
- CO<sub>2</sub> and gas (air) is not an option
- Different liquids proposed as options with its advantages and disadvantages
- At 297-298K temperature homogeneity on the sensor
- PGW6040 proposed as solution for 2,15 m/s reducing the material budget and obtaining a homogeneous T along the sensor

